



# ΕΘΝΙΚΟ ΜΕΤΣΟΒΙΟ ΠΟΛΥΤΕΧΝΕΙΟ ΣΧΟΛΗ ΜΗΧΑΝΟΛΟΓΩΝ ΜΗΧΑΝΙΚΩΝ

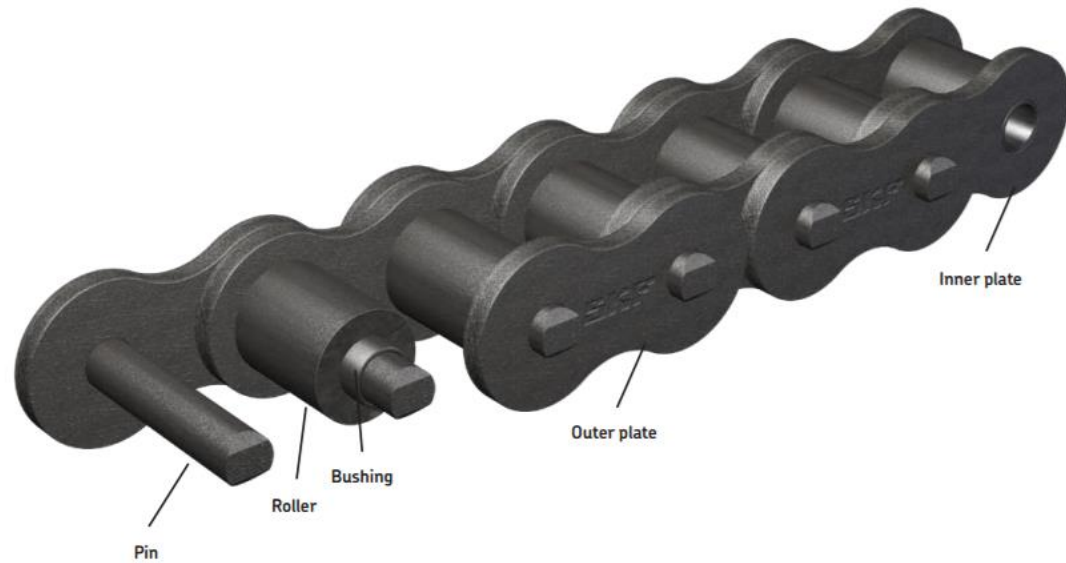
ΕΞΑΜΗΝΟ 4<sup>ο</sup>  
ΣΤΟΙΧΕΙΑ ΜΗΧΑΝΩΝ II

## ΑΛΥΣΟΚΙΝΗΣΕΙΣ

14/05/2020

ΓΕΩΡΓΙΟΣ ΒΑΣΙΛΕΙΟΥ, Διπλ. Μηχανολόγος Μηχανικός ΕΜΠ  
ΚΩΝΣΤΑΝΤΙΝΟΣ ΤΣΙΟΥΜΑΝΗΣ ΜΑΝΟΣ, Διπλ. Μηχανολόγος Μηχανικός ΕΜΠ

# ΣΤΟΙΧΕΙΑ ΜΗΧΑΝΩΝ ΑΛΥΣΟΚΙΝΗΣΕΩΝ



# ΤΥΠΟΙ ΑΛΥΣΙΔΩΝ



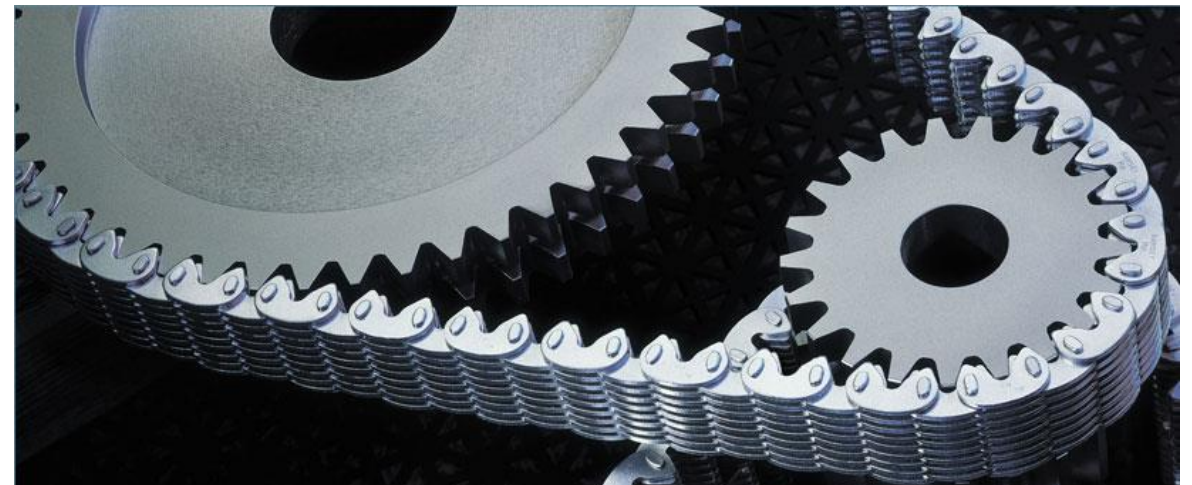
Roller chains – most common use



Double pitch chains – conveyors



Engineered drive chains – high strength

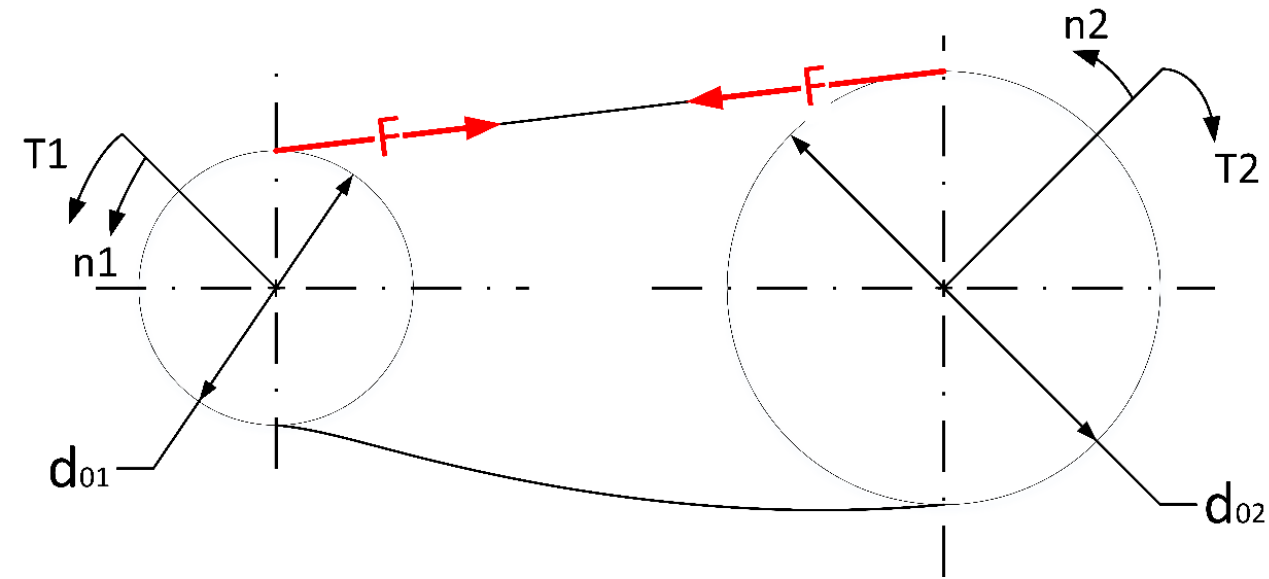


Silent chains – low noise, tooth shaped links

# ΠΡΟΣΘΕΤΑ ΣΤΟΙΧΕΙΑ ΜΗΧΑΝΩΝ



# ΚΙΝΗΜΑΤΙΚΗ ΚΑΙ ΔΥΝΑΜΙΚΗ



- Δυνάμεις:

$$F = \frac{T_1}{\frac{d_{01}}{2}} = \frac{T_2}{\frac{d_{02}}{2}}$$

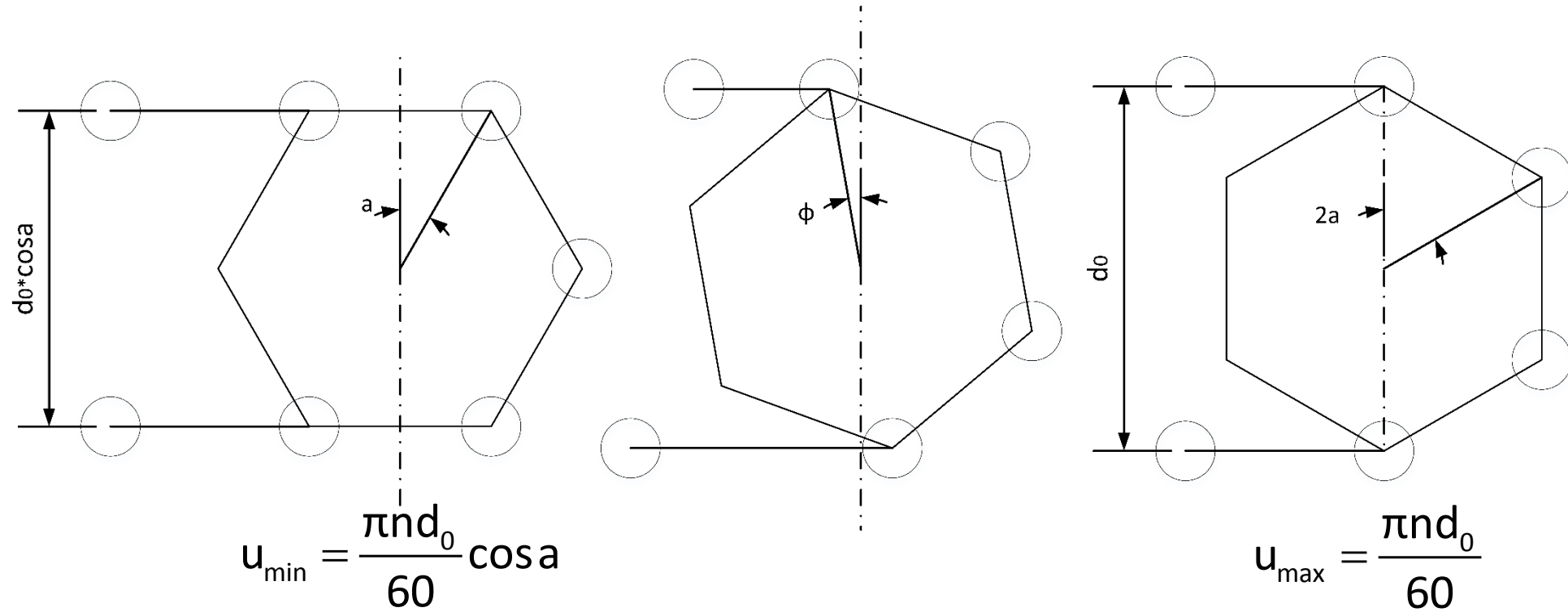
- Ταχύτητα περιστροφής:

$$v = w_1 \frac{d_{01}}{2} = w_2 \frac{d_{02}}{2} \Leftrightarrow \frac{n_1 \pi d_{01}}{60} = \frac{n_2 \pi d_{02}}{60}$$

- Σχέση μετάδοσης:

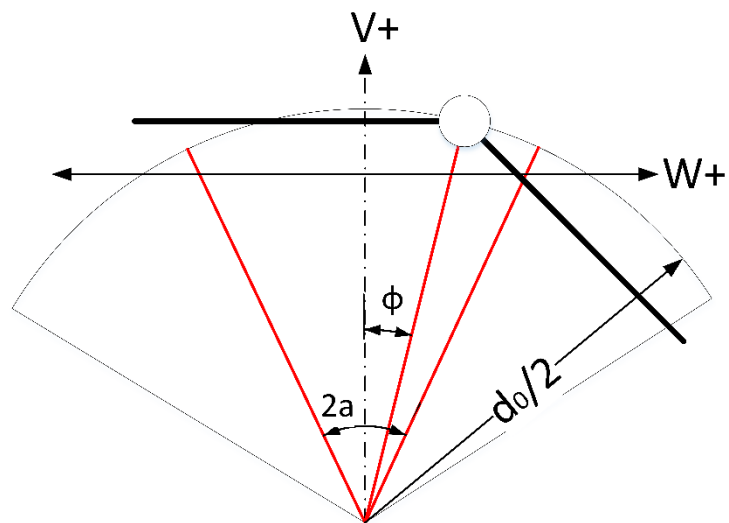
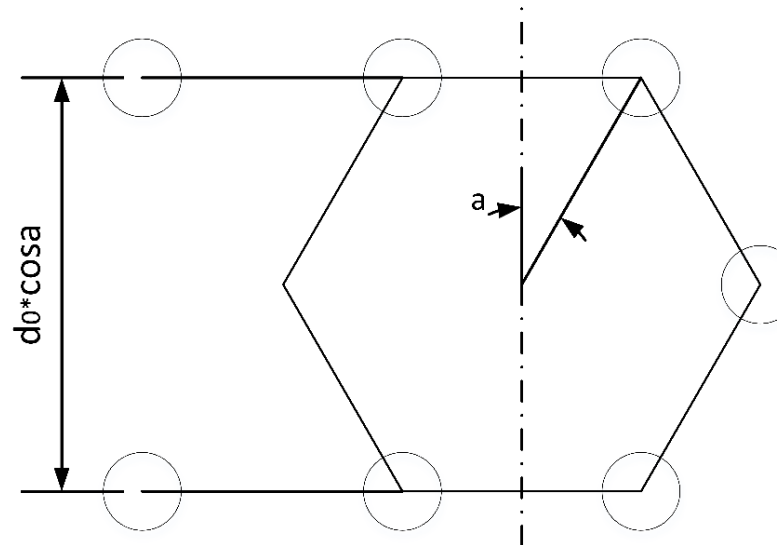
$$i_{12} = \frac{z_2}{z_1}$$

# ΦΑΙΝΟΜΕΝΟ ΠΟΛΥΓΩΝΟΥ



$$\left. \begin{array}{l} n_{\text{sprocket}} = \text{constant} \\ t_{\text{chain}} = t_{\text{sprocket}} \end{array} \right\} i_{12} = \text{constant?}$$

# ΦΑΙΝΟΜΕΝΟ ΠΟΛΥΓΩΝΟΥ



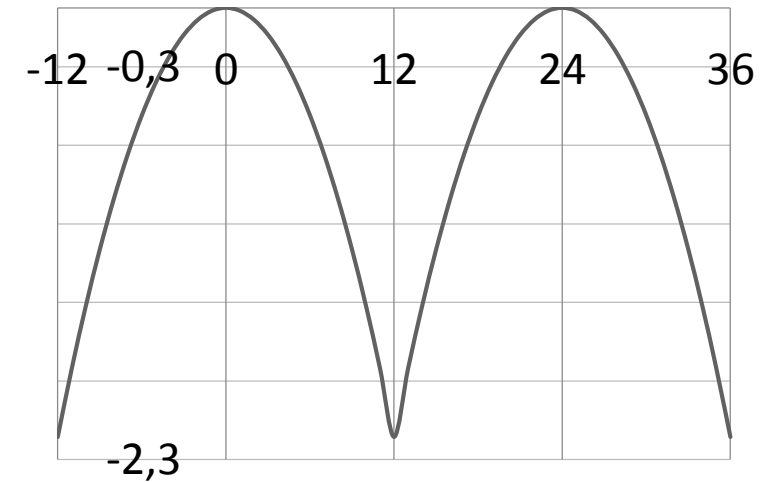
$$W_1 = \frac{d_{01}}{2} \sin \varphi_1, -a \leq \varphi \leq a$$

$$V_1 = \frac{dW_1}{dt} = \frac{d_{01}}{2} \cos \varphi_1 \frac{d\varphi_1}{dt}$$

$$t = d_{01} \sin \alpha_1$$

$$i_{\text{real}} = \frac{\frac{d_{02}}{2} \cos \varphi_2}{\frac{d_{01}}{2} \cos \varphi_1} = \frac{\sin \alpha_1 \cos \varphi_2}{\sin \alpha_2 \cos \varphi_1}$$

$\Delta u$  (%)



# ΦΑΙΝΟΜΕΝΟ ΠΟΛΥΓΩΝΟΥ

Διακύμανση σχέσης μετάδοσης

$$\Delta i_{\min} = \frac{z_1 \sin a_1}{z_2 \sin a_2} \left( \frac{\cos a_2}{\cos a_1} - 1 \right)$$

$$\Delta i_{\max} = \frac{z_1 \sin a_1}{z_2 \cos a_1 \sin a_2} (1 - \cos a_1 \cos a_2)$$



# ΔΥΝΑΜΙΚΗ ΑΛΥΣΟΚΙΝΗΣΕΩΣ

## ΕΓΚΑΡΣΙΕΣ ΤΑΛΑΝΤΩΣΕΙΣ

Βαθμοί Ελευθερίας:  $m = 2 \left( \frac{l_k}{t} - 1 \right) + 2$

$$m_{tr} \ddot{y}_r = \frac{F}{t} (y_{r+1} - 2y_r + y_{r-1})$$

$$m_t [\text{kg}] = m_{\text{length}} \left[ \frac{\text{kg}}{\text{m}} \right] t [\text{m}]$$

$$f_E = \frac{1}{\pi} \sqrt{\frac{F}{m_t t}} \sin\left(\frac{\lambda}{\rho+1} \frac{\pi}{2}\right) \xrightarrow{?} f_E = \frac{1}{2} \frac{\lambda}{l_k} \sqrt{\frac{F}{m_{\text{length}}}}$$

Φαινόμενο πολυγώνου:

$$f_d = v \frac{z_1 n_1}{60} = v \frac{z_2 n_2}{60}$$

$$f_d = f_E \Rightarrow n_{\text{critical}} = 30 \frac{\lambda}{l_k} \frac{1}{v z_1} \sqrt{\frac{F}{m_{\text{length}}}} = 30 \frac{\lambda}{l_k} \frac{1}{v z_1} \sqrt{\frac{2T_1}{m_{\text{length}} d_{01}}}$$

# ΔΥΝΑΜΙΚΗ ΑΛΥΣΟΚΙΝΗΣΕΩΣ

## ΔΙΑΜΗΚΕΙΣ ΤΑΛΑΝΤΩΣΕΙΣ

Βαθμοί Ελευθερίας:  $m = 2 \left( \frac{l_k}{t} - 1 \right) + 2$

$$m_{tr} \ddot{x}_r = k_t (x_{r+1} - 2x_r + x_{r-1})$$

$$m_t [\text{kg}] = m_{\text{length}} \left[ \frac{\text{kg}}{\text{m}} \right] t [\text{m}]$$

$$f_E = \frac{1}{\pi} \sqrt{\frac{k_t}{m_t}} \sin\left(\frac{\lambda}{\rho+1} \frac{\pi}{2}\right) \xrightarrow{?} f_E = \frac{1}{2} \frac{\lambda}{l_k} \sqrt{\frac{k_t t}{m_{\text{length}}}}$$

$$k' = \frac{k_t t}{F_{\text{Fail}}}$$

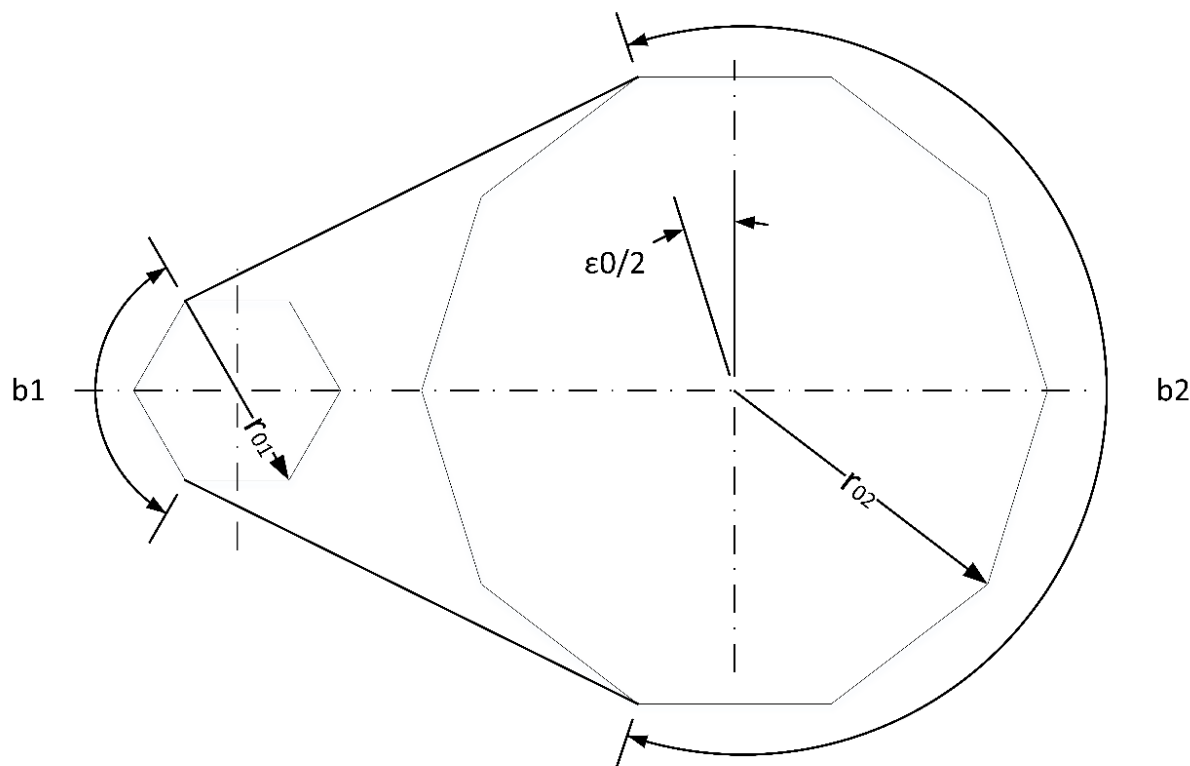
$$f_E = \frac{1}{2} \frac{\lambda}{l_k} \sqrt{\frac{k' F_{\text{Fail}}}{m_{\text{length}}}}$$

Φαινόμενο πολυγώνου:

$$f_d = v \frac{z_1 n_1}{60} = v \frac{z_2 n_2}{60}$$

$$f_d = f_E \Rightarrow n_{\text{critical}} = 30 \frac{\lambda}{l_k} \frac{1}{v z_1} \sqrt{\frac{k' F_{\text{Fail}}}{m_{\text{length}}}}$$

# ΓΕΩΜΕΤΡΙΑ ΑΛΥΣΟΚΙΝΗΣΕΩΣ



## DIN 8195

$$X = 2 \frac{a_0}{t} + \frac{z_1 + z_2}{2} + \left( \frac{z_2 - z_1}{2\pi} \right)^2 \frac{t}{a_0}$$

$$a_0 = \frac{t}{4} \left[ \left( X - \frac{z_1 + z_2}{2} \right) + \sqrt{\left( X - \frac{z_1 + z_2}{2} \right)^2 - 2 \left( \frac{z_2 - z_1}{\pi} \right)^2} \right]$$

## Μέθοδος Worobjew

$$X = \frac{2a_0 \cos \varepsilon_0}{t} + \frac{z_1 + z_2}{2} + \frac{\varepsilon_0 (z_2 - z_1)}{180}$$

$$a_0 = \frac{t}{2 \cos \varepsilon_0} \left( X - \frac{z_1 + z_2}{2} + \frac{\varepsilon_0 (z_2 - z_1)}{180} \right)$$

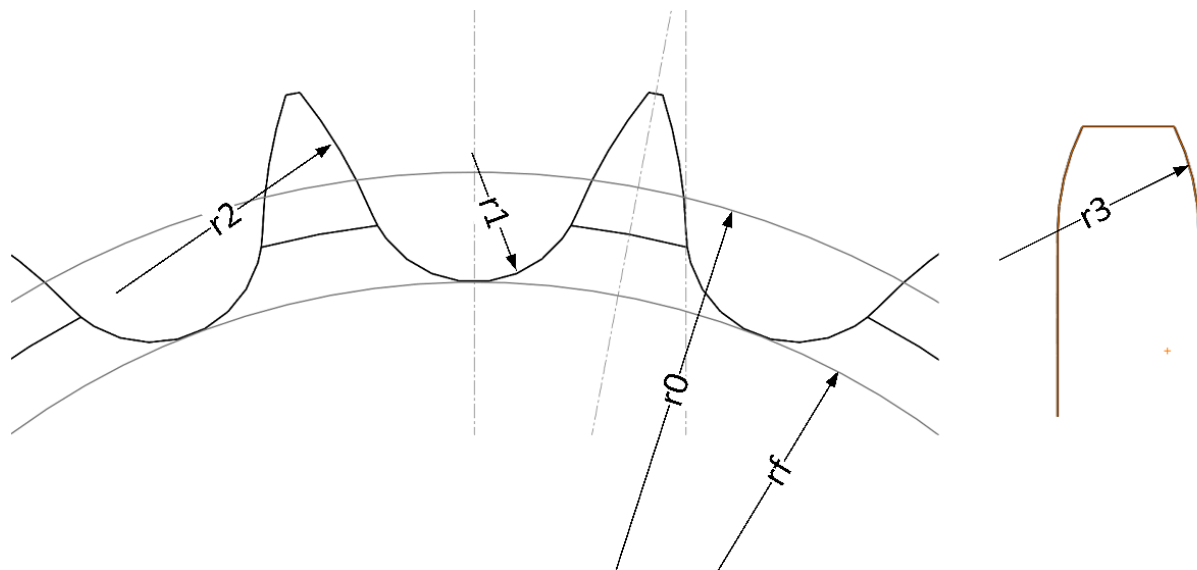
## Βιομηχανία

$$L = \frac{z_1 + z_2}{2} + 2C + \frac{K}{C}$$

$$C = 30 \div 80t$$

$$K : \text{tables} \propto z_2 - z_1$$

# ΓΕΩΜΕΤΡΙΑ ΟΔΟΝΤΩΣΕΩΝ



$$d_0 = \frac{t}{\sin a}$$

$$a = \frac{180^\circ}{z}$$

$$d_f = d_0 - d_1$$

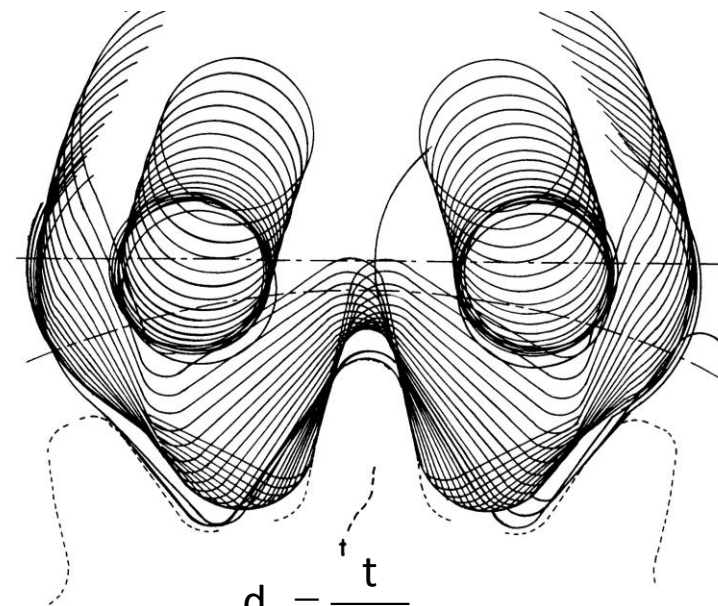
$$d_k = d_0 \cos a + 2k$$

$$r_1 = (0.51 \pm 0.005) d_1$$

$$r_2 = \begin{cases} 0.12d_1(z+2), \text{max} \\ 0.008(z^2+180), \text{min} \end{cases}$$

$$r_3 = 1.5d_1$$

$$\gamma = 15 \div 19^\circ$$



$$d_0 = \frac{t}{\sin a}$$

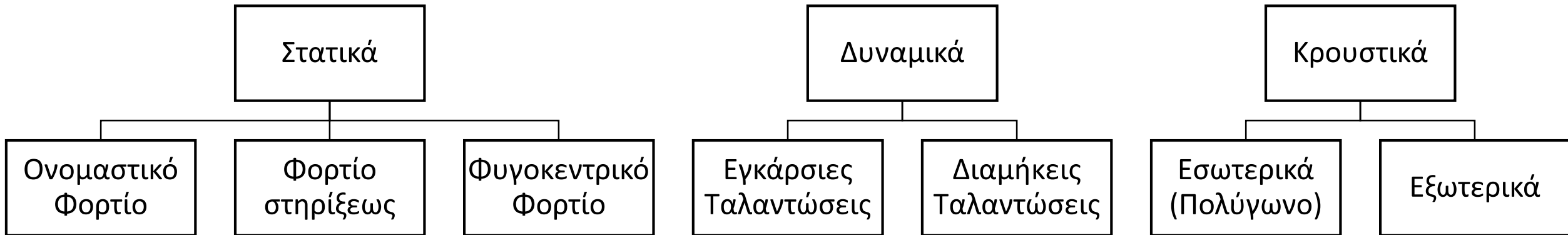
$$a = \frac{180^\circ}{z}$$

$$d_f = d_0 - h_1 = d_k - h_2$$

$$d_k = d_0 \cos a$$

$$\gamma = 30^\circ - 2a$$

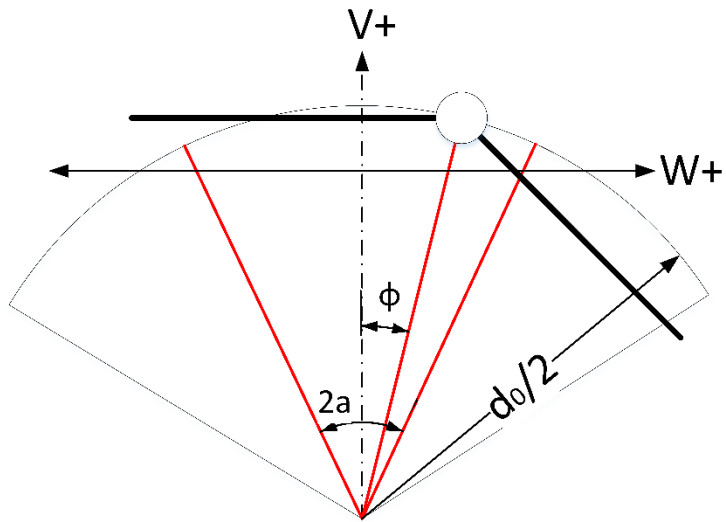
# ΦΟΡΤΙΑ ΑΛΥΣΟΚΙΝΗΣΕΩΣ



# ΣΤΑΤΙΚΑ ΦΟΡΤΙΑ

- Ονομαστικό φορτίο  $F = \frac{T_1}{d_{01}/2} = \frac{T_2}{d_{02}/2}$

$$F = \frac{T_1}{d_{01}/2} = \frac{2T_1}{d_{01} \cos \phi}$$



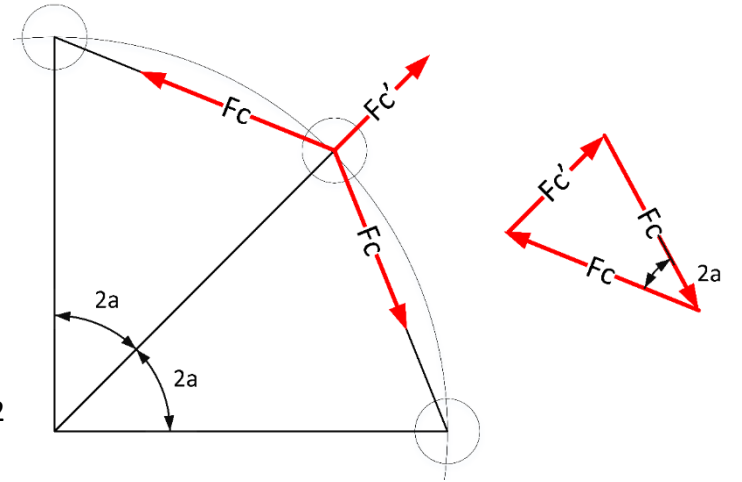
- Φυγοκεντρικό φορτίο

$$F_C' = m_{\text{length}} t \frac{v^2}{r}$$

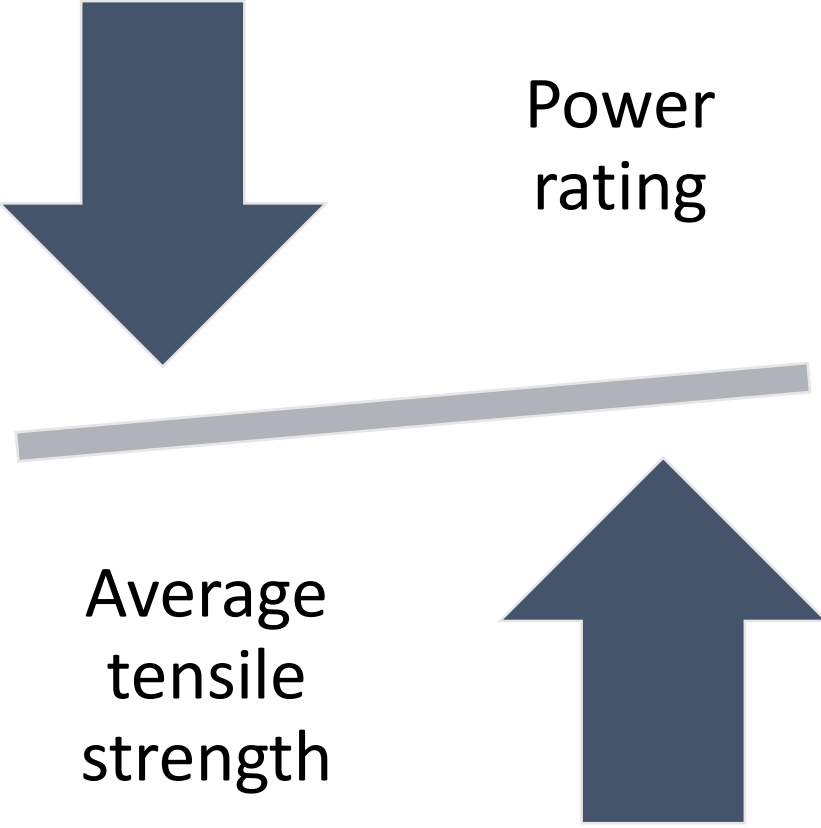
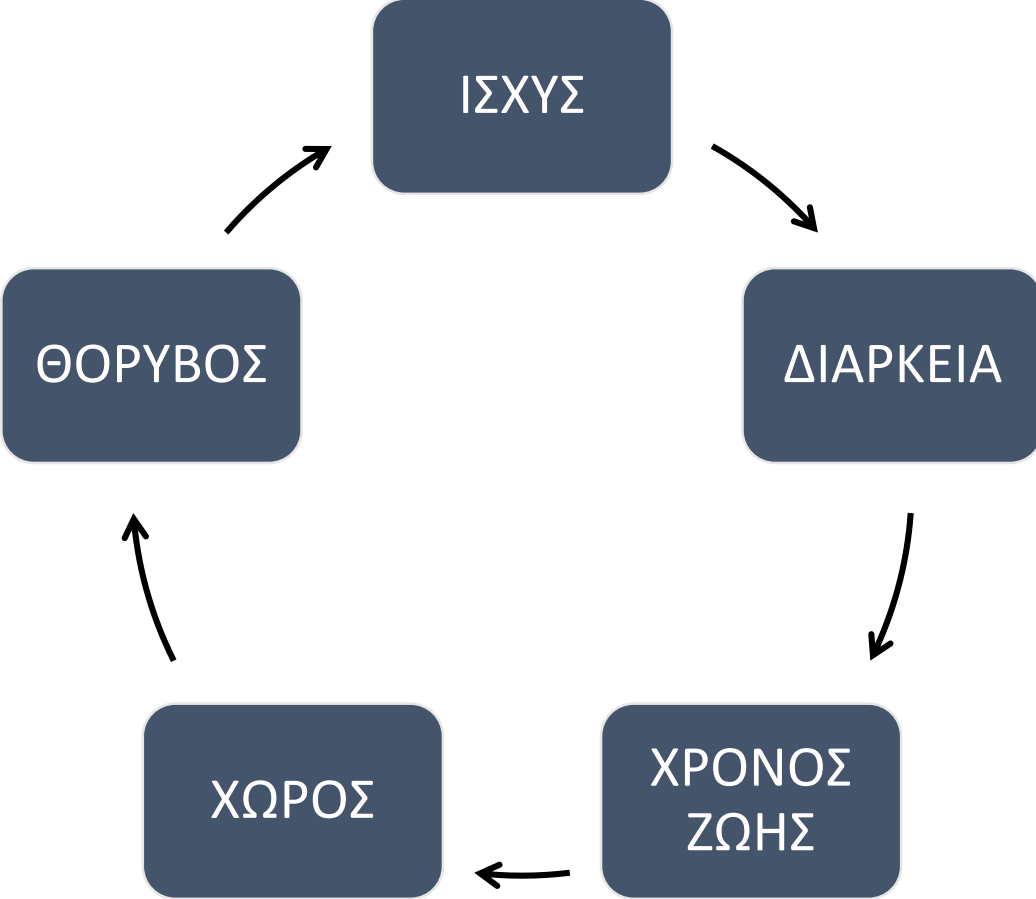
$$r = \frac{d_0}{2} = \frac{t}{\sin a}$$

$$F_C' = m_{\text{length}} v^2 \sin a$$

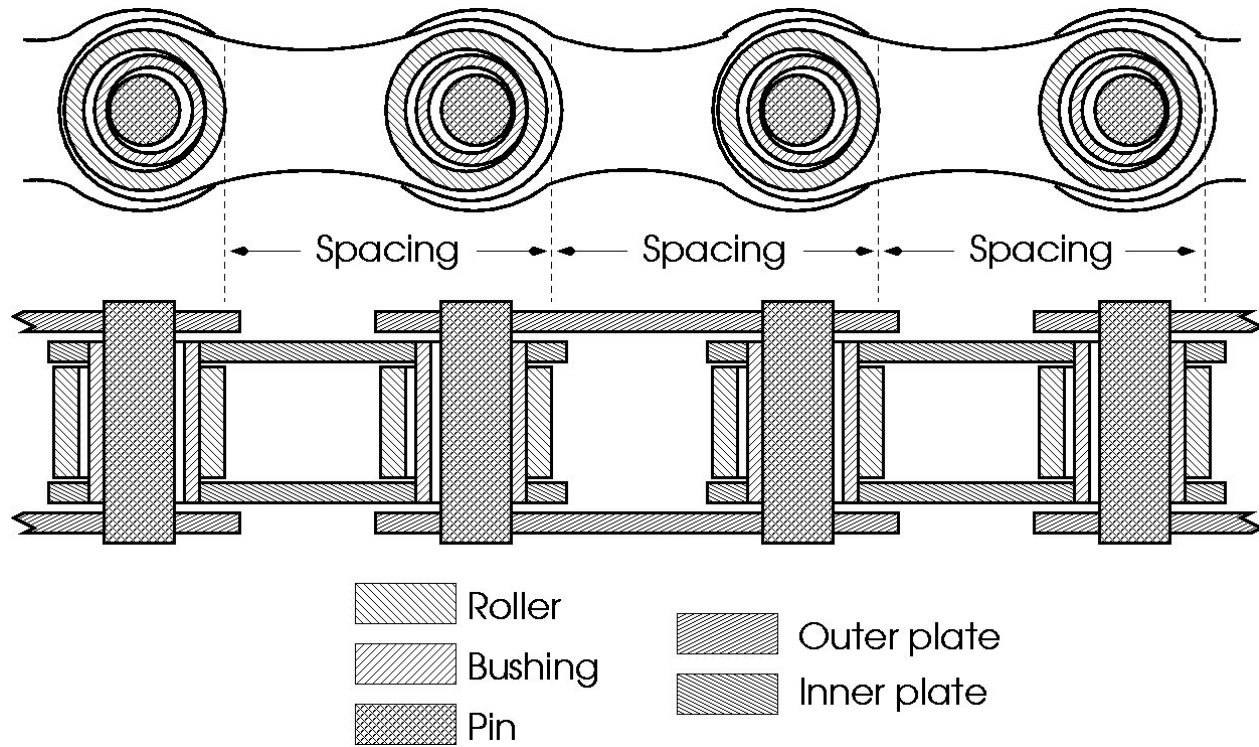
$$F_C = \frac{F_C'}{\sin a} = m_{\text{length}} v^2$$



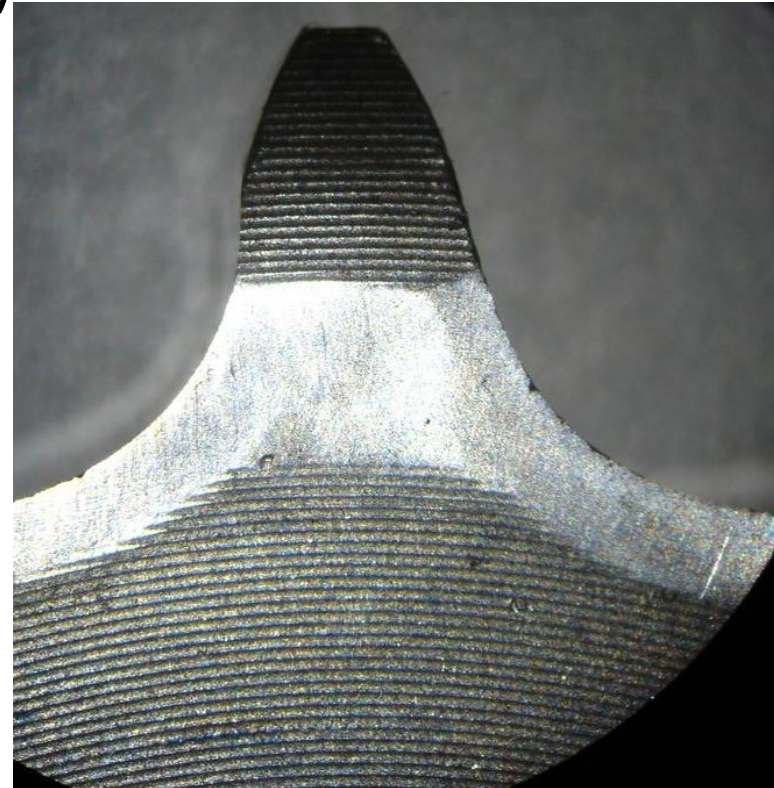
# ΕΠΙΛΟΓΗ ΑΛΥΣΙΔΑΣ



# ΦΘΟΡΑ ΚΑΙ ΔΙΑΡΚΕΙΑ ΖΩΗΣ



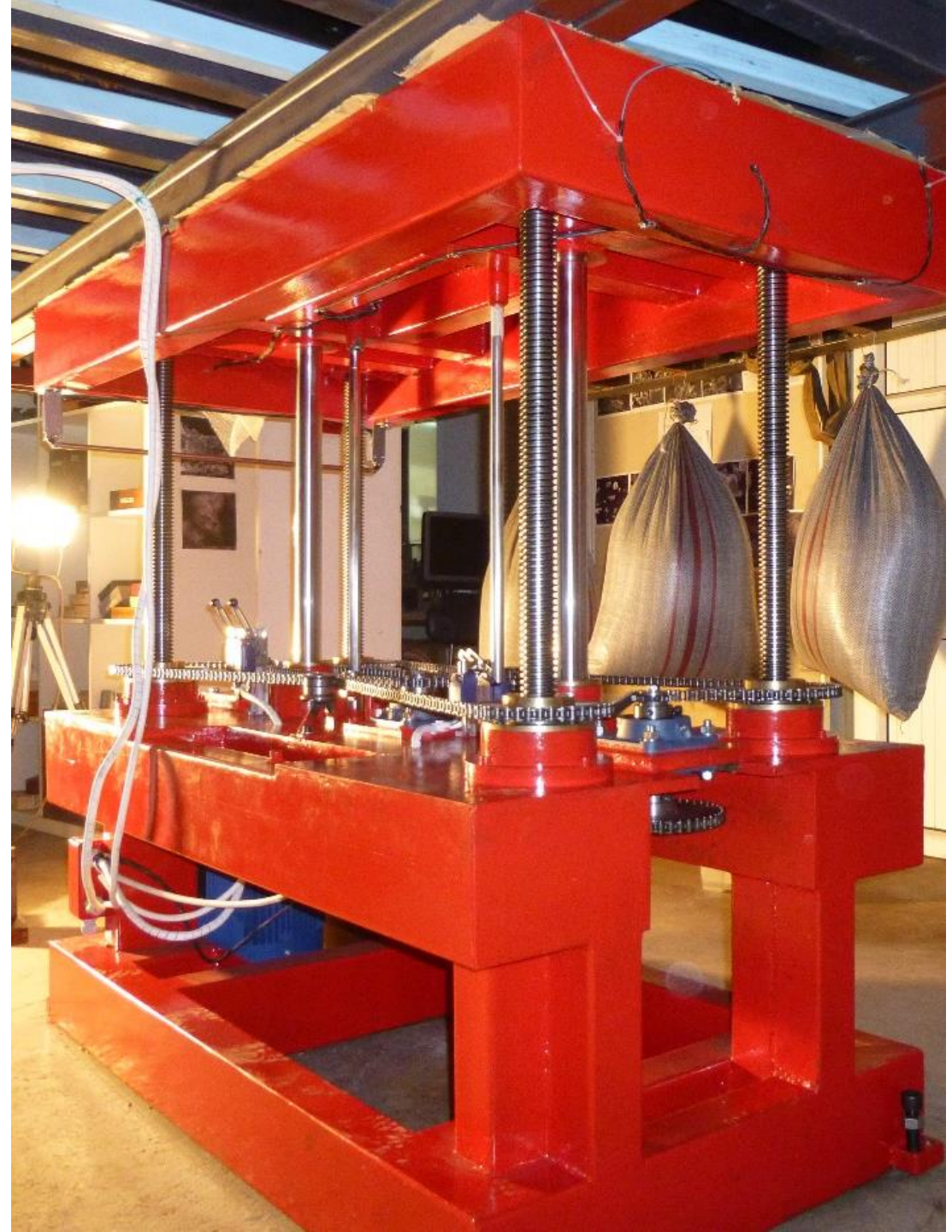
Νόμος φθοράς πίεσης επιφανείας – ταχύτητας ολίσθησης (p-v wear rule)



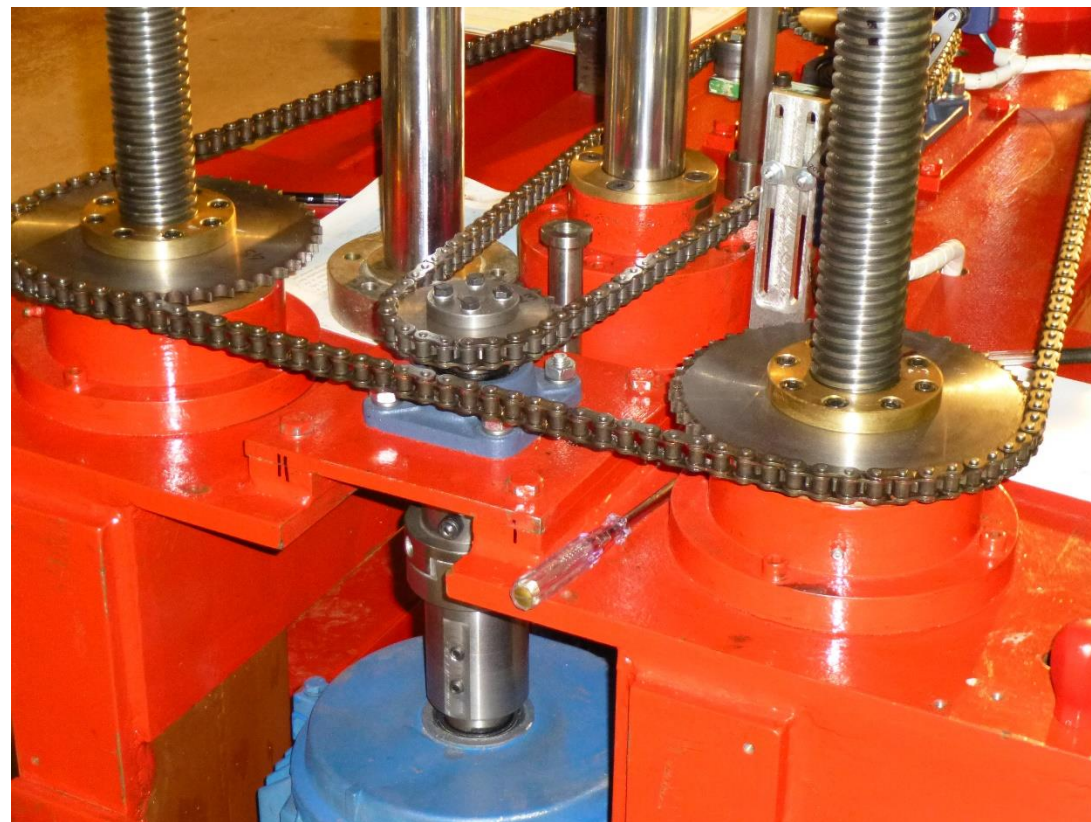
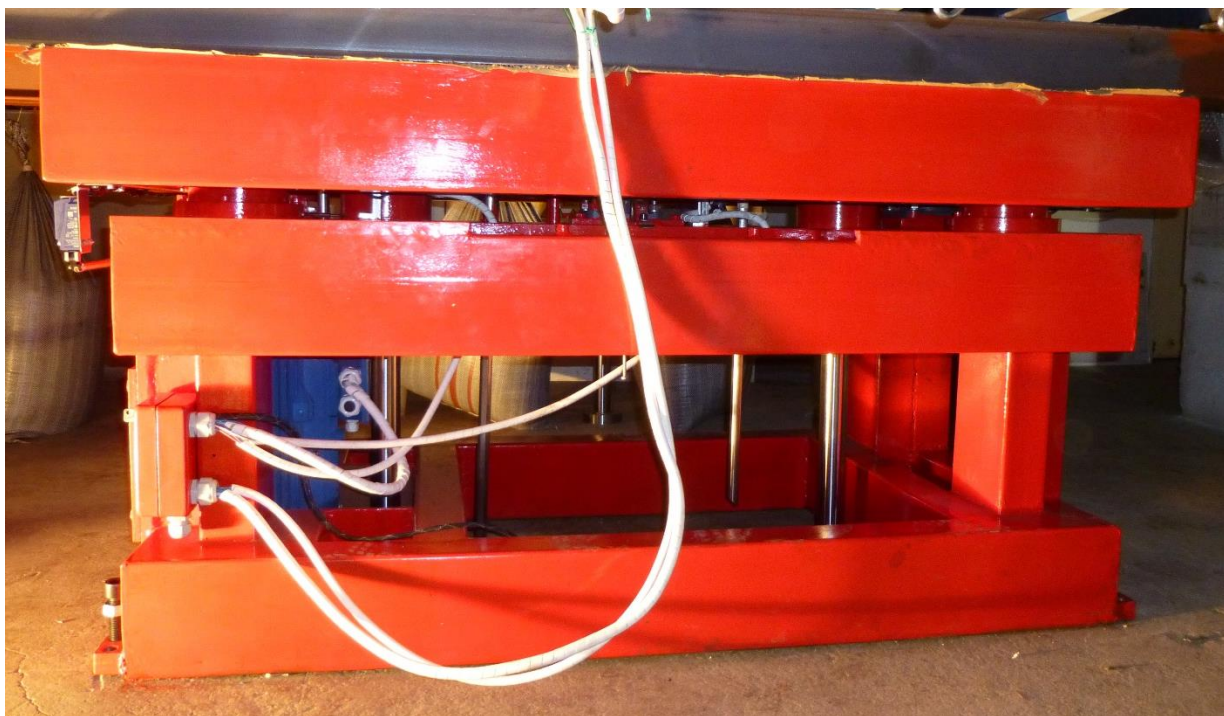


# ΕΦΑΡΜΟΓΗ

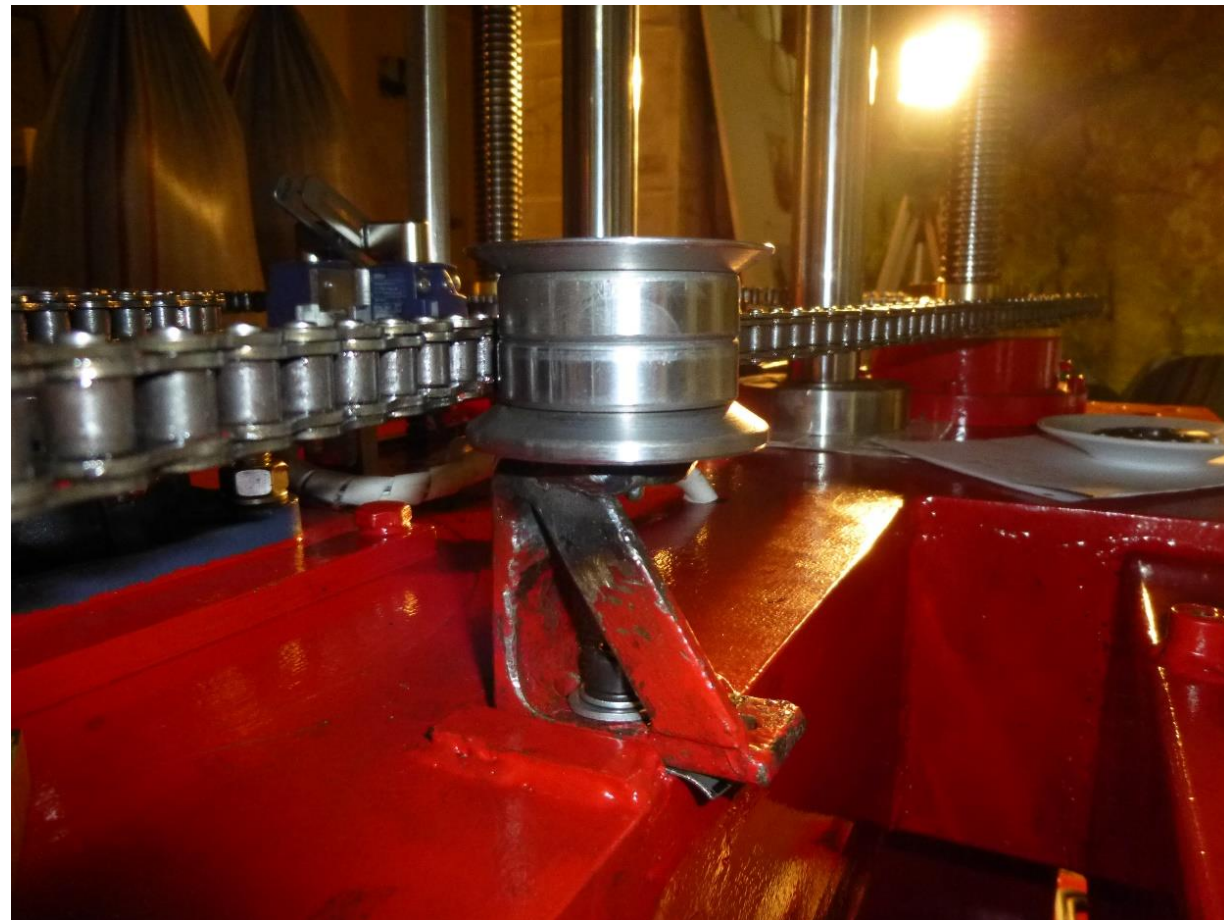
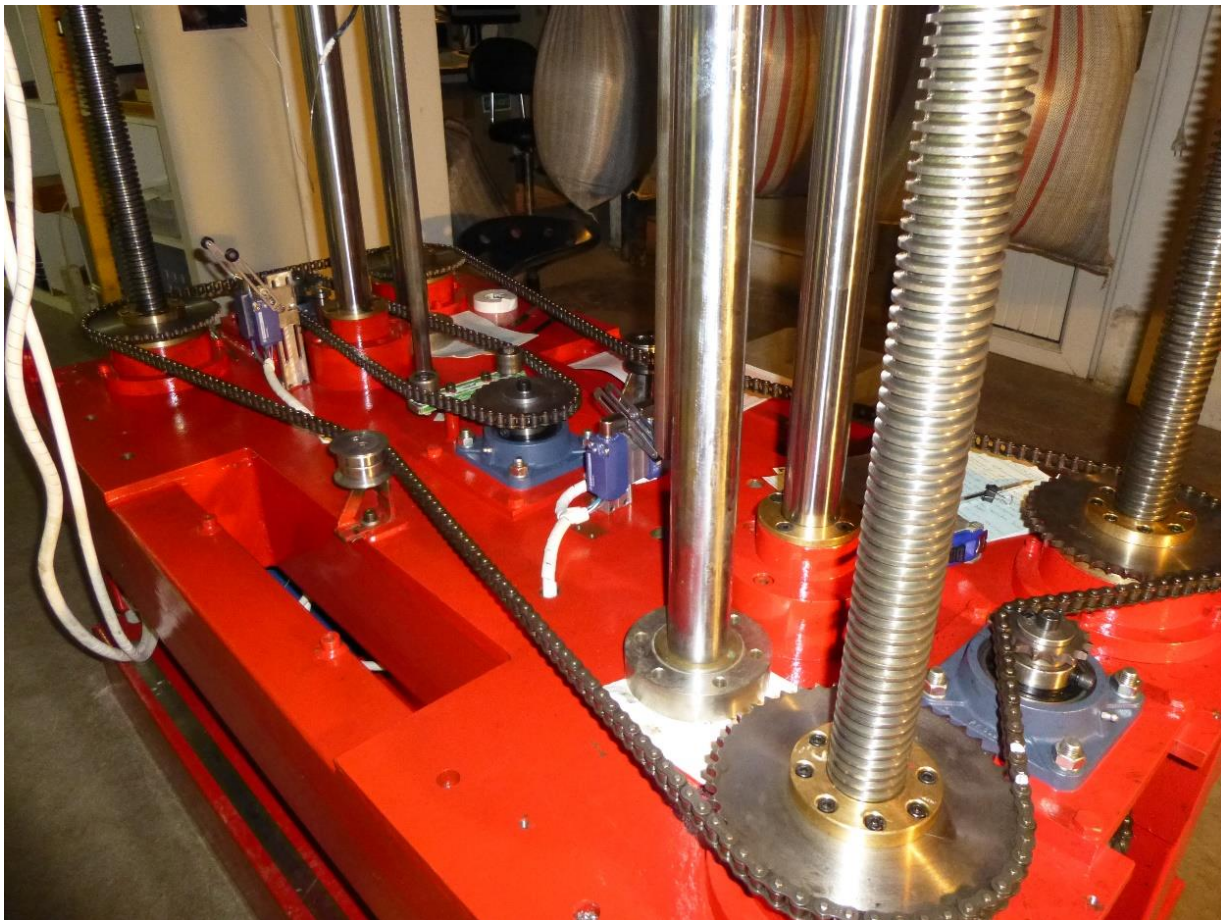
- Four remotely controlled vertical lifting platforms (10MT) for the XTDV - CERN
- Chain Drive
- 3 speed reduction stages



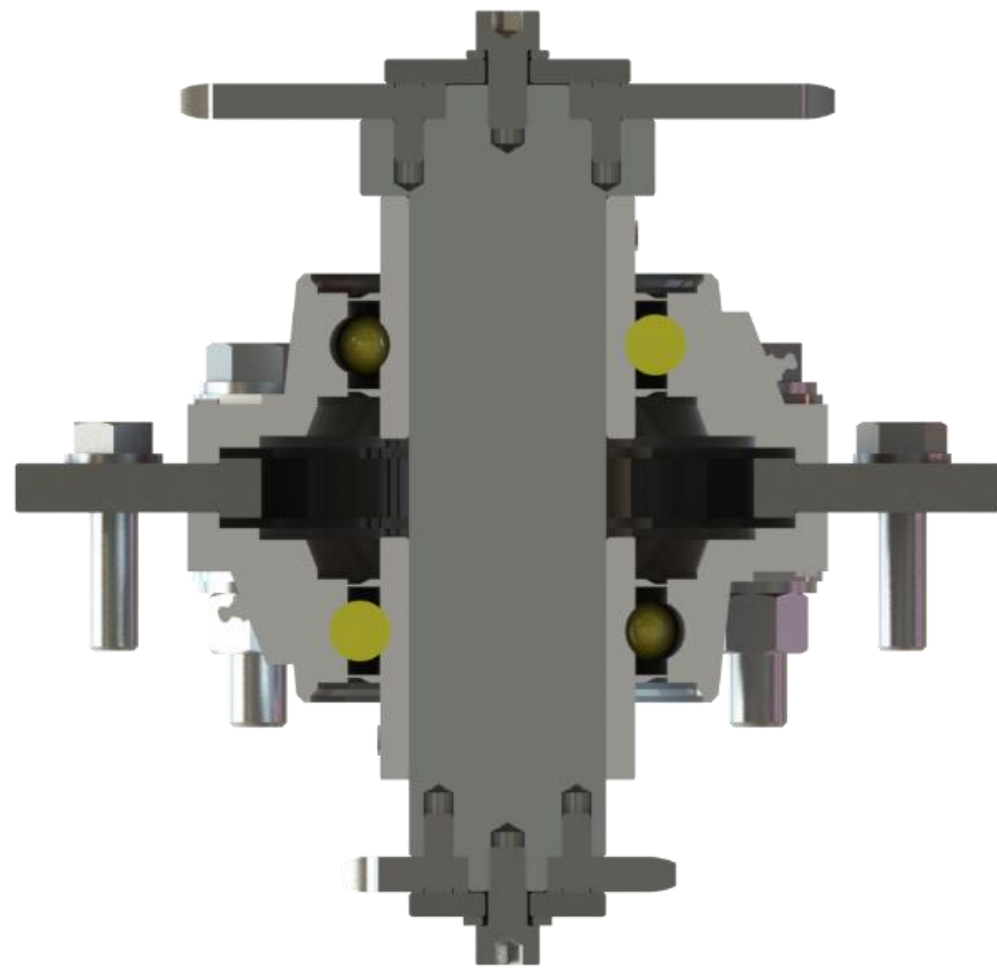
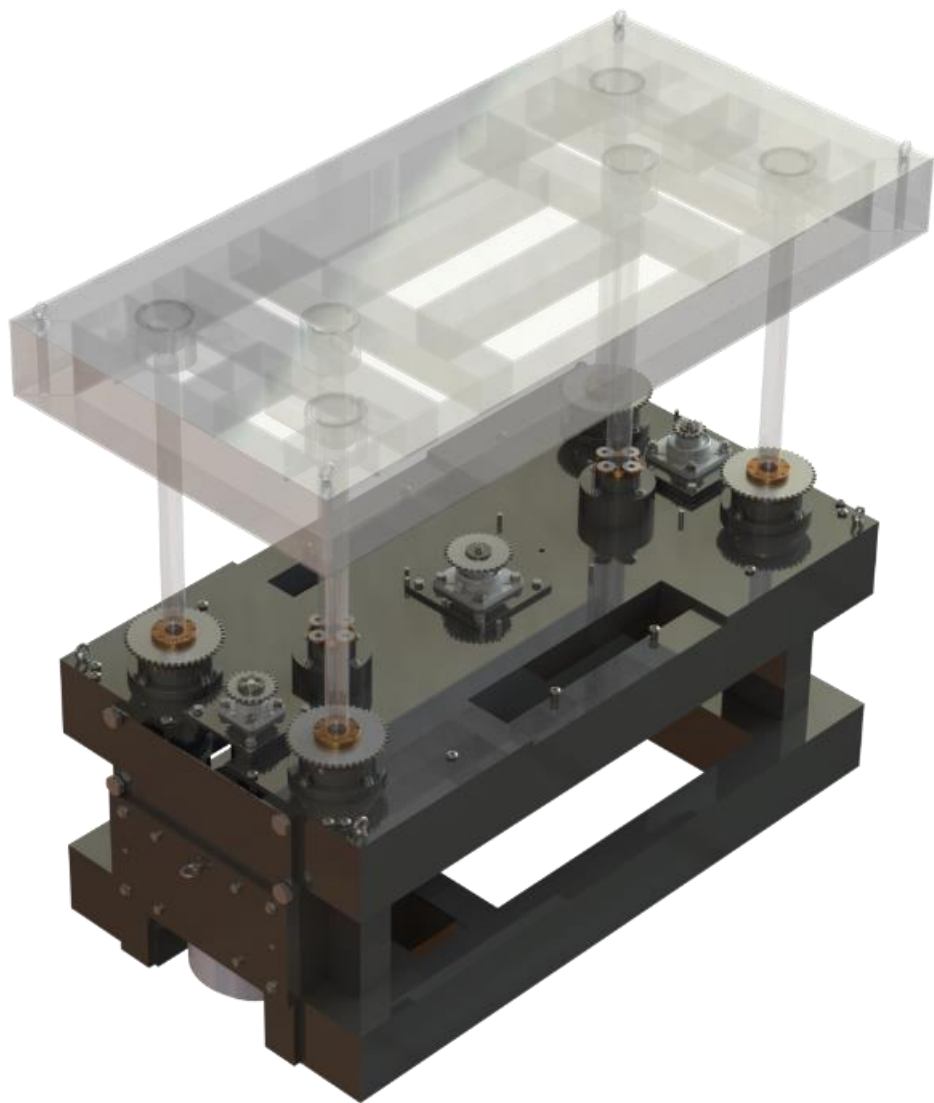
# ΕΦΑΡΜΟΓΗ



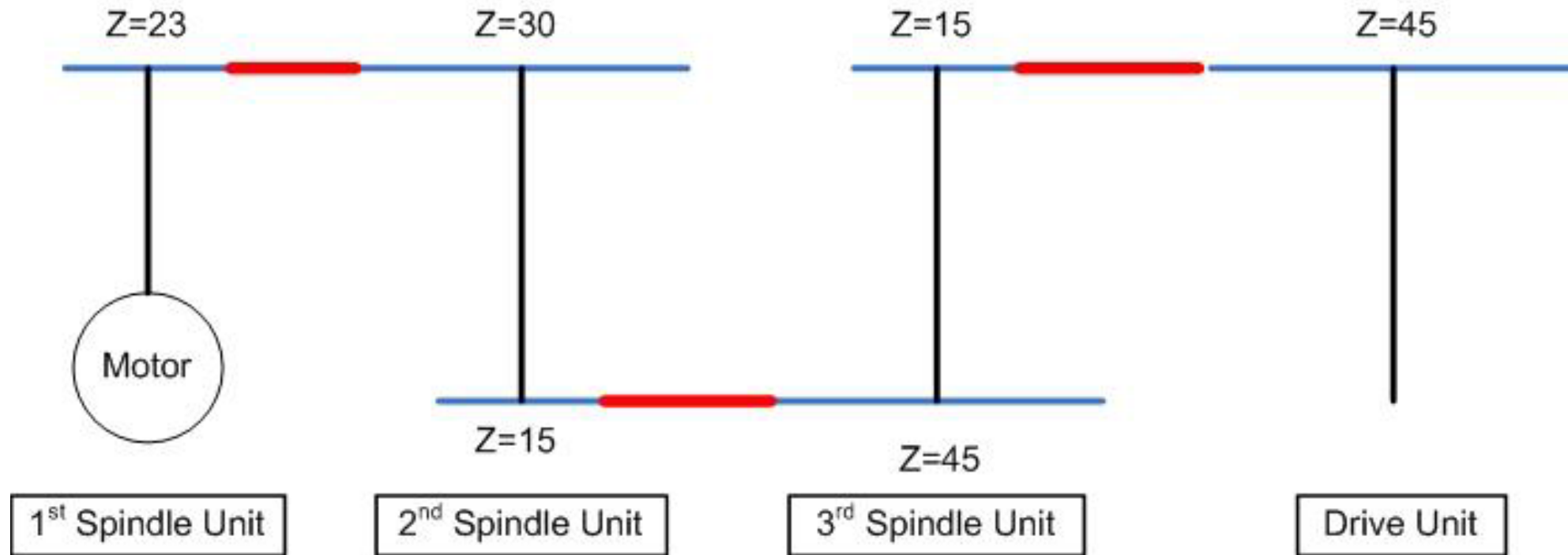
# ΕΦΑΡΜΟΓΗ



# ΕΦΑΡΜΟΓΗ



# Chain Drive



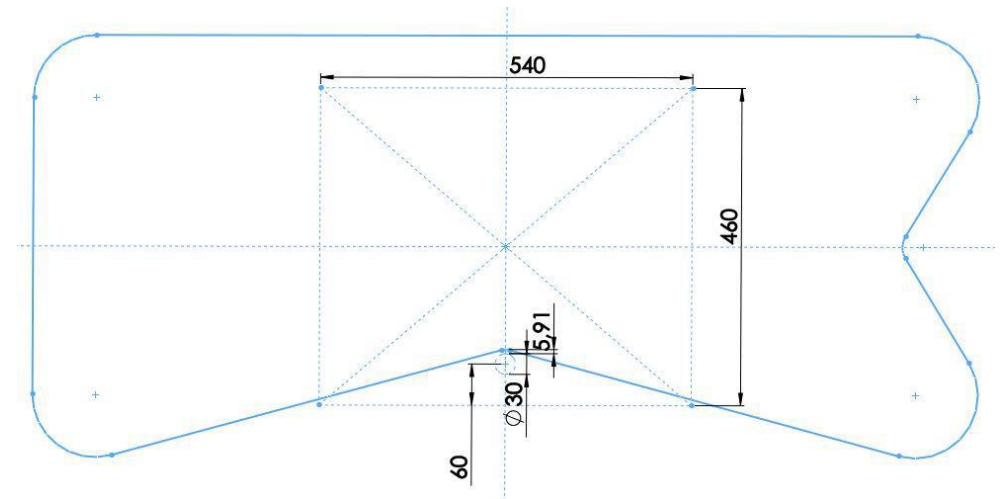
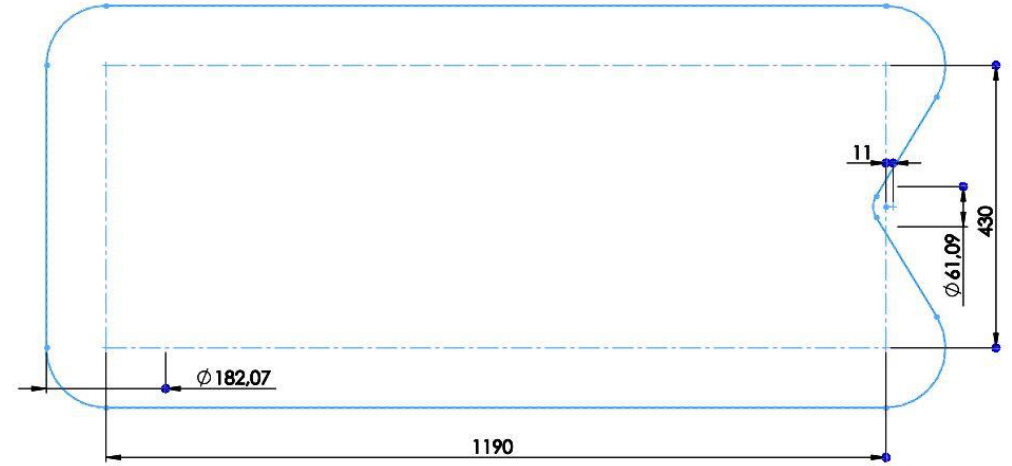
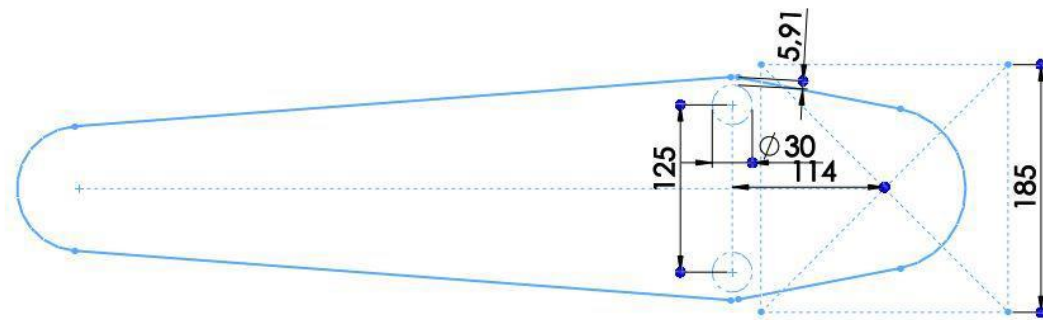
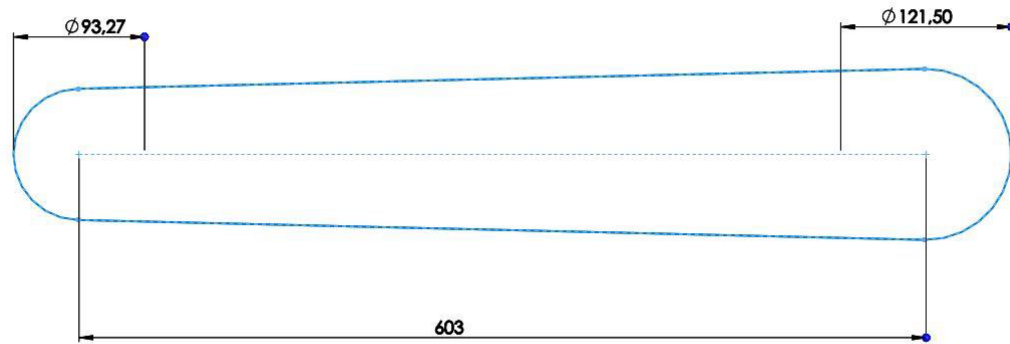
- 3 stage speed reduction
- Speed reduction ratio  $i=11,74$
- Chain 08B-1 (1/2")
- Tensile Strength 21,1kN

# Chain Drive



- 3 stage speed reduction
- Speed reduction ratio  $i=11,74$
- Chain 08B-1 (1/2")
- Tensile Strength 21,1kN

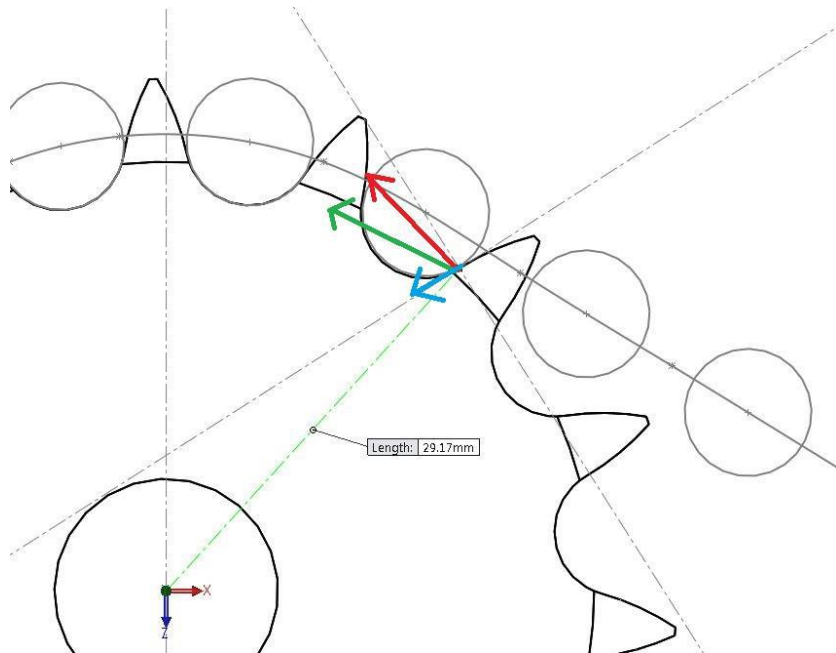
# Chain Drive - Tensioners



# ΥΠΟΛΟΓΙΣΜΟΣ ΑΛΥΣΙΔΑΣ

- ΑΝΤΟΧΗ

Εφελκυσμός:  $F = \frac{M}{d_p / 2}$



- ΦΘΟΡΑ

Νόμος πίεσης-ταχύτητας ολίσθησης (p-v wear rule)

$$p_{\text{pin}} = \frac{M}{d_{p,\text{sprocket}} / 2} \frac{1}{A_{\text{pin}}}$$

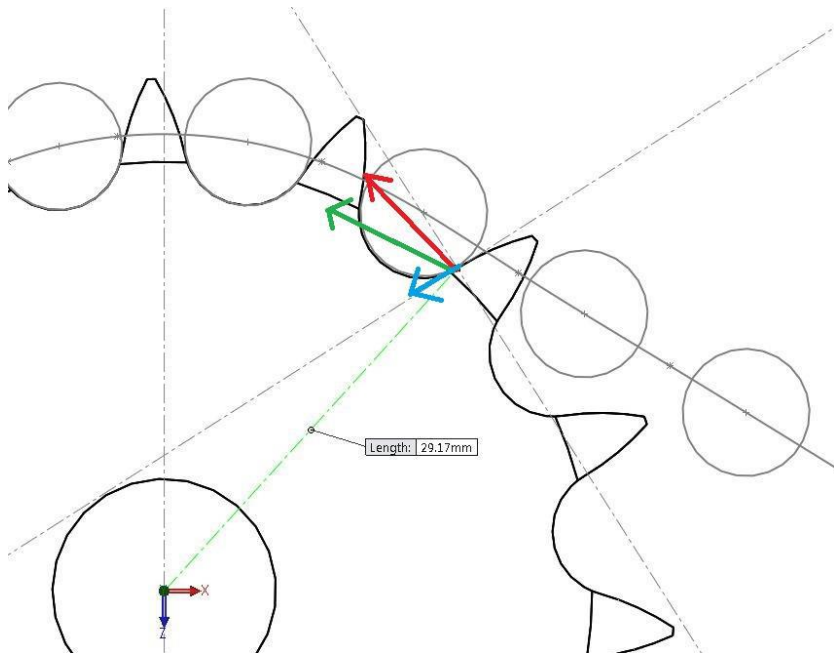
$$\frac{dh}{dt} = kp v_{\text{contact}}$$



# ΥΠΟΛΟΓΙΣΜΟΣ ΑΛΥΣΙΔΑΣ

## ΦΘΟΡΑ

Νόμος πίεσης-ταχύτητας ολίσθησης (p-v wear rule)



$$p_{\text{pin}} = \frac{M}{d_{p,\text{sprocket}} / 2} \frac{1}{A_{\text{pin}}}$$

$$\frac{dh}{dt} = kp v_{\text{contact}}$$

$$z = 15$$

$$d_0 = 61.09\text{mm}$$

$$\omega = 36.13 \text{ rad/s}$$

$$r_{\text{pin}} = 8.51\text{mm}$$

$$b = 7.2\text{mm}$$

$$T = 130\text{Nm}$$

$$r_{\text{wheel}} = 29.17\text{mm}$$

$$r_{\text{pin}} = 30.545$$

$$\theta_{\text{wheel}} = 74.79^\circ$$

$$\theta_{\text{pin}} = 65.84^\circ$$

$$k = 1.23 * 10^{-15} \frac{\text{m}^3}{\text{Nm}}$$

$$\theta_{\text{slip}} = 0.33^\circ$$

$$t_{\text{stroke}} = 40\text{s}$$

# CERN 10MT LIFTING PLATFORMS

